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## **Introducing Anisotropic Eddy-viscosity Coefficient with Single-equation Model**

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### **Abstract**

A one-equation turbulence model is formulated which retains an anisotropic eddy viscosity coefficient. Consequently, the current model is deemed to be potential for accounting near-wall turbulence and strong flow in-homogeneity, enhancing the predictive accuracy for complex separated and reattaching flows. Furthermore, the devised turbulence model retrieves the link to the  $k$ - $\varepsilon$  model and is likely to be extendable toward a non-linear algebraic Reynold stress model. Intuitively, the current artifact may accommodate a variable eddy-viscosity coefficient for an LES (large eddy simulation) or a DES (Detached eddy simulation) method.

**Keywords:** one-equation model, near-wall turbulence, non-equilibrium flow, LES, DES.

### **1 Introduction**

To replicate both equilibrium and non-equilibrium flow in one-equation models, considerable innovative research has been undertaken [1–7]. Empiricism and arguments of dimensional analysis are involved in the widely used one-equation Spalart and Allmaras (SA) model [1], avoiding the link to the traditional  $k$ - $\varepsilon$  turbulence model. Internal and external flows are extensively utilized to calibrate and validate the SA model; providing reasonable predictions. However, connection to the  $k$ - $\varepsilon$  model has been ameliorated with recently developed one-equation models by Rahman et al. [3–6]. They reproduce relatively improved predictions for separated and reattaching flows owing to their capability in accounting for non-equilibrium

effects via variable model coefficients. In fact, a single-equation turbulence model establishes a good compromise between algebraic and two-equation models because of inheriting transport effects.

Customarily, one/two-equation models encounter non-equilibrium effects when being embedded with an anisotropic eddy-viscosity coefficient  $C_\mu$ , parameterized with a production to dissipation ratio  $P_k/\varepsilon$  accompanied by invariants of mean strain-rate and vorticity tensors. The resulting  $C_\mu$  suppresses non-physical energy components at moderate/severe strain rates on the perspective of realizability constraint, representing a minimal requirement for the turbulence model. Therefore, the current eddy-viscosity formulation with an appropriate strain-dependent  $C_\mu$  reinforces turbulence anisotropy in a single-equation model. In addition,  $k$  and  $\varepsilon$  are explored in the present model, reviving presumably the competency in speculating complex separated and reattaching flows.

## 2 Formulation of present turbulence model

A transport equation for  $R = C_\mu k^2/\varepsilon$  (pseudo-eddy viscosity) can be obtained using the two-equation  $k$ - $\varepsilon$  turbulence model. The following relation is used to construct an  $R$ -transport equation:

$$\frac{DR}{Dt} = \frac{D(C_\mu k^2/\varepsilon)}{Dt} = C_\mu \left( \frac{2k}{\varepsilon} \frac{Dk}{Dt} - \frac{k^2}{\varepsilon^2} \frac{D\varepsilon}{Dt} \right) = \frac{2R}{k} \frac{Dk}{Dt} - \frac{R}{\varepsilon} \frac{D\varepsilon}{Dt} \quad (1)$$

where the substantial derivative is indicated by  $D/Dt$ . Equations of  $k$  and  $\varepsilon$  at a high Reynolds number can be provided with:

$$\frac{Dk}{Dt} = P_k - \varepsilon + \nabla \cdot \left( \frac{R}{\sigma_k} \nabla k \right) \quad (2)$$

$$\frac{D\varepsilon}{Dt} = C_{\varepsilon 1} \frac{\varepsilon}{k} P_k - C_{\varepsilon 2} \frac{\varepsilon^2}{k} + \nabla \cdot \left( \frac{R}{\sigma_\varepsilon} \nabla \varepsilon \right) \quad (3)$$

where  $P_k$  implies the production term; relevant model constants are  $\sigma_k, \sigma_\varepsilon, C_{\varepsilon 1}$  and  $C_{\varepsilon 2}$ . Combining Equations (1)-(3) and carrying out some algebra with  $\sigma_k = \sigma_\varepsilon = \sigma_R$ , result in an  $R$ -transport equation:

$$\begin{aligned} \frac{DR}{Dt} = & (2 - C_{\varepsilon 1}) \frac{R}{k} P_k - (2 - C_{\varepsilon 2}) k + \frac{\partial}{\partial x_j} \left[ \left( \frac{R}{\sigma_R} \right) \frac{\partial R}{\partial x_j} \right] - \frac{2R^2}{k^2 \sigma_R} \frac{\partial k}{\partial x_j} \frac{\partial k}{\partial x_j} \\ & + \frac{4R}{k \sigma_R} \frac{\partial k}{\partial x_j} \frac{\partial R}{\partial x_j} - \frac{2}{\sigma_R} \frac{\partial R}{\partial x_j} \frac{\partial R}{\partial x_j} \end{aligned} \quad (4)$$

Apparently, the diffusion/destruction term  $\left( \frac{4R}{k \sigma_R} \frac{\partial k}{\partial x_j} \frac{\partial R}{\partial x_j} - \frac{2R^2}{k^2 \sigma_R} \frac{\partial k}{\partial x_j} \frac{\partial k}{\partial x_j} \right)$  appearing in the above-mentioned relation may be excluded in order to avoid the

numerical stiffness. Therefore, Equation (4) can be regularized using the Bradshaw relation  $|-uv| = \sqrt{C_\mu} k = R \left| \frac{du}{dy} \right|$  [8] with the  $k$ - $\varepsilon$  source and sink terms:

$$\frac{D\rho R}{Dt} = C_1 \rho R \tilde{S} - C_\mu^* \rho k / 4 + \frac{\partial}{\partial x_j} \left[ \left( \mu + \frac{\mu_T}{\sigma_R} \right) \frac{\partial R}{\partial x_j} \right] - C_2 \rho \frac{\partial R}{\partial x_j} \frac{\partial R}{\partial x_j} \quad (5)$$

where  $C_1 = \sqrt{C_\mu (C_{\varepsilon 2} - C_{\varepsilon 1})}$ ,  $C_{\varepsilon 1} = 1.44$ ,  $C_{\varepsilon 2} = 1.9 + C_\mu / 4$ ,  $C_\mu = 0.09$ ,  $C_2 = 2/\sigma_R$ , and  $C_{\varepsilon 1} = 1.44$ ,  $C_{\varepsilon 2} = 1.9 + C_\mu / 4$ ,  $C_\mu = 0.09$ ,  $C_2 = 2/\sigma_R$  and  $\sigma_R = 1.3$ . The Park-Park limiter [9] is applied to determine the eddy-viscosity  $\mu_T$ :

$$\mu_T = \rho \min \left[ \sqrt{\frac{2}{3}} \frac{k T_t}{\zeta}; \min(f_{v1} R; C_\mu^* k T_t) \right] \quad (6)$$

where the hybrid time scale  $T_t$  can be given by [5]:

$$T_t = \sqrt{\frac{k^2}{\varepsilon^2} + C_T^2 \frac{\nu}{\varepsilon}} = \frac{k}{\varepsilon} \sqrt{1 + \frac{C_T^2}{Re_T}}, \quad Re_T = \frac{k^2}{\nu \varepsilon} \quad (7)$$

where  $Re_T$  denotes the turbulent Reynolds number,  $C_T = \sqrt{2}$  is an empirical constant and  $\nu (= \mu/\rho; \rho$  is the density and  $\mu$  is the molecular viscosity) signifies the kinematic viscosity.

The mean strain-rate  $S_{ij}$  and vorticity  $W_{ij}$  tensors, required afterward can be defined By

$$S_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right), \quad W_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right) \quad (8)$$

The invariants of vorticity and mean strain-rate tensors can be represented by  $W = \sqrt{2W_{ij}W_{ij}}$  and  $S = \sqrt{2S_{ij}S_{ij}}$ , respectively. The eddy-viscosity coefficient  $C_\mu^*$  in Equation (6) has been formed with mean strain-rate and vorticity invariants;  $C_\mu^*$  as suggested in Reference [5] has been adopted:

$$C_\mu^* = \frac{\alpha_1}{1 - \frac{3}{2}\eta^2 + 2\xi^2} \quad (9)$$

where non-dimensional mean shear strain-rate and mean rotation-rate parameters are defined by  $\eta = \alpha_2 T_t S$  and  $\xi = \alpha_3 T_t W$ , respectively. Coefficients  $\alpha_1$ - $\alpha_3$  in Equation (9) are given by:

$$\alpha_1 = g \left( \frac{1}{4} + \frac{2}{3} \sqrt{\Pi_b} \right), \quad \alpha_2 = \frac{3}{8\sqrt{2}} g \quad (10)$$

$$\alpha_3 = \frac{3}{\sqrt{2}} \alpha_2, \quad g = \left( 1 + 2 \frac{P_k}{\varepsilon} \right)^{-1}$$

with  $\Pi_b = b_{ij}b_{ij}$ ; the Reynolds stress anisotropy  $b_{ij}$  is characterized by

$$b_{ij} = \frac{\overline{u_i u_j}}{2k} - \frac{1}{3} \delta_{ij} \quad (11)$$

Rahman et al. [3–6] have devised three explicit solutions to  $P_k/\varepsilon$ . The current formulation utilizes the simplest one, indicating a universal value of  $\sqrt{\Pi_b} \approx 0.31$  in homogeneous turbulence. In addition,  $P_k/\varepsilon$  receives an approximated consistency condition with  $P_k/\varepsilon = \zeta/3.2$  in homogeneous turbulence, where  $\zeta = T_t S \max(1, \mathfrak{R})$ , and  $\mathfrak{R} = |W/S|$  indicates a non-dimensional variable [5].

The associated quantity  $f_{v1}$  in Equation (6) represents an eddy-damping function, defined by

$$f_{v1} = \left[ 1 - \exp\left(-\frac{y}{L}\right) \right]^2, \quad L^2 = \zeta (7.1 + C_\mu^* \text{Re}_T) \sqrt{\frac{v^3}{\varepsilon}} \quad (12)$$

where  $y$  implies a wall-distance parameter usually normal to the wall and  $(v^3/\varepsilon)^{1/4}$  signifies the Kolmogorov length scale.

The tensor  $\tilde{S}$  linked to the production term in Equation (5) represents the scalar measure of deformation; unlike the  $SA$  turbulence model [1], this term is redefined in order to take the effect of vorticity into account:

$$\tilde{S} = f_k \left( S - \frac{|\eta_1| - \eta_1}{2} \right), \quad f_k = 1 - \frac{f_{v1}}{2} \sqrt{\max(1 - \mathfrak{R}, 0)} \quad (13)$$

where  $\eta_1 = S - W$ . Equation (13) has a close resemblance to an enhancement approach for the turbulence model sensitivity toward the effect of streamline curvature, provoking an extra rate of strain over and above the main strain-rate in the flow field. The kinetic energy of turbulence  $k$  can be approximated with the aid of Bradshaw's relation as [11]:

$$k = f_{v1}^{0.3} \frac{RS_k}{\sqrt{C_\mu}}, \quad S_k = \sqrt{\tilde{S}^2 + S_\alpha^2} \quad (14)$$

The non-vanishing strain-rate correction term  $S_\alpha$  in the free-stream region can be designed using the log-law behaviour of pseudo-eddy viscosity  $R = u_\tau k y$  and  $du/dy = u_\tau/k y$  as:

$$S_\alpha = f_{v1} \max \left[ \left( \frac{3}{2\kappa} \frac{\partial \sqrt{R}}{\partial x_i} \right)^2; \frac{1}{C_\mu} s^{-1} \right] \quad (15)$$

where  $1/C_\mu s^{-1}$  has been estimated from a nearly homogeneous shear flow [10] and the von-Karman constant  $\kappa = 0.41$ . The hybrid time scale  $T_t$  requires an evaluation of the total dissipation-rate  $\varepsilon$  since it plays an important role in constructing a compatible  $T_t$ ;  $\varepsilon$  is determined as follows [11]:

$$\varepsilon = \sqrt{\varepsilon_\omega^2 + \tilde{\varepsilon}^2}, \quad \tilde{\varepsilon} = RS_k^2 f_{v1}^{1.3} \quad (16)$$

where unlike  $\varepsilon$ ,  $\tilde{\varepsilon}$  disappears at the solid wall due to the product  $(R \times f_{v1}^{1.3})$ . However,  $\varepsilon_w$  indicates the wall-dissipation rate, balanced by the viscous-diffusion rate at the wall vicinity;  $\varepsilon_w$  is conventionally modelled as:

$$\varepsilon_w = 2A_\varepsilon \nu \left( \frac{\partial u}{\partial y} \right)_w^2 \approx 2A_\varepsilon \nu S_k^2 \quad (17)$$

where  $A_\varepsilon = C_\mu = 0.09$  from *DNS* data. Apparently, the total dissipation-rate  $\varepsilon$  is likely to be benefited by the wall dissipation-rate  $\varepsilon_w$  within the wall-layer.

### 3 Results

Fully-developed turbulent channel flows at  $Re_\tau = 180, 395$  and  $640$  are simulated to substantiate the model accuracy in replicating near-wall turbulence. Computations are carried out in a half-width  $h$  of the channel using a *1-D* (one-dimensional) *RANS* solver. A non-uniform  $1 \times 64$  grid resolution for  $Re_\tau = (180; 395)$  and  $1 \times 128$  grid resolution for  $Re_\tau = 640$  are assumed to be adequate to accurately describe characteristics of the flow. To assure the viscous sublayer resolution, the first near-wall grid spacing is set to  $y^+ \approx 0.3$ . A cell-centered finite-volume approach is applied to solve the flow equations. Results are converted to the form of  $u^+ = u/u_\tau$ ,  $k^+ = k/u_\tau^2$ ,  $\overline{uv}^+ = \overline{uv}/u_\tau^2$ ,  $\varepsilon^+ = \nu\varepsilon/u_\tau^4$ , where  $u_\tau$  is the wall-friction velocity; comparisons are made by plotting these quantities versus  $y^+ = yu_\tau/\nu$ . Turbulence quantities are extracted from *DNS* data [12, 13]. Predictions of the present model are compared with those of the widely-used *SA* turbulence model [1].

The stream-wise mean  $x$ -momentum equation for a *1-D* incompressible flow can be represented by

$$\frac{\partial p}{\partial x} + \frac{\partial}{\partial y} \left[ (v + v_T) \frac{\partial u}{\partial y} \right] = 0 \quad (18)$$

where  $v_T = \mu_T/\rho$  is the kinematic eddy-viscosity; lower and upper wall locations of the channel are encompassed by  $y = (-h; h)$ . The axial pressure-gradient  $\partial p/\partial x$  remains constant and the continuity constraint  $\partial u_i/\partial x_i = 0$  is naturally satisfied, as the mean flow field has a *1-D* feature. However,  $\partial p/\partial x$  must be computed as a part of the solution method since the pressure gradient is not known a priori. The pressure-velocity correction (*PVC*) method [14, 15] is an appropriate choice to solve the problem. The *PVC* scheme keeps updating the axial pressure gradient and velocity as long as the fictitious mass source is minimized.

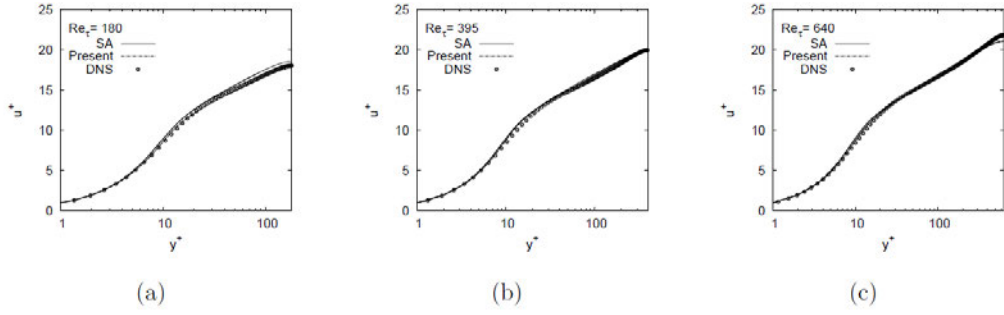


Figure 1: Velocity profiles for fully-developed turbulent channel flow.

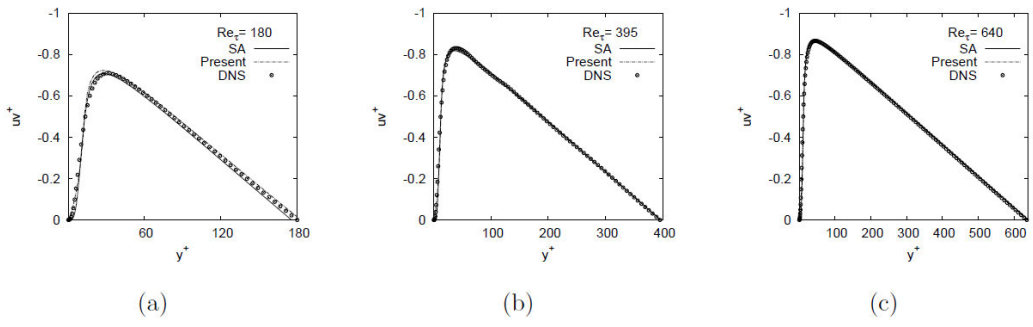


Figure 2: Shear stress profiles for fully-developed turbulent channel flow.

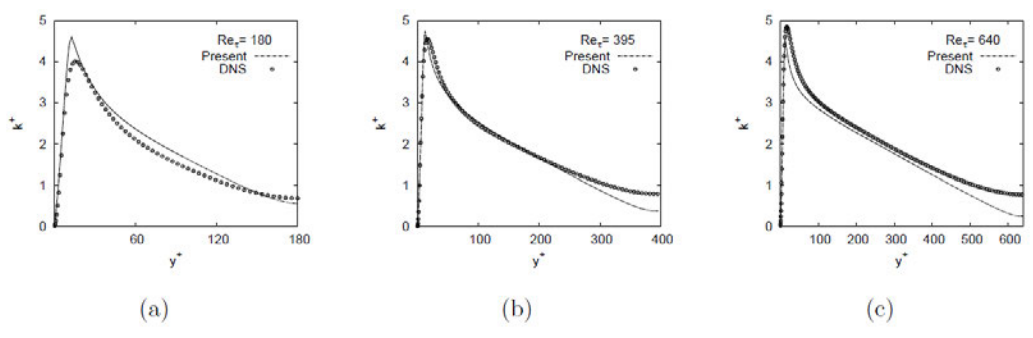


Figure 3: Kinetic energy profiles for fully-developed turbulent channel flow.

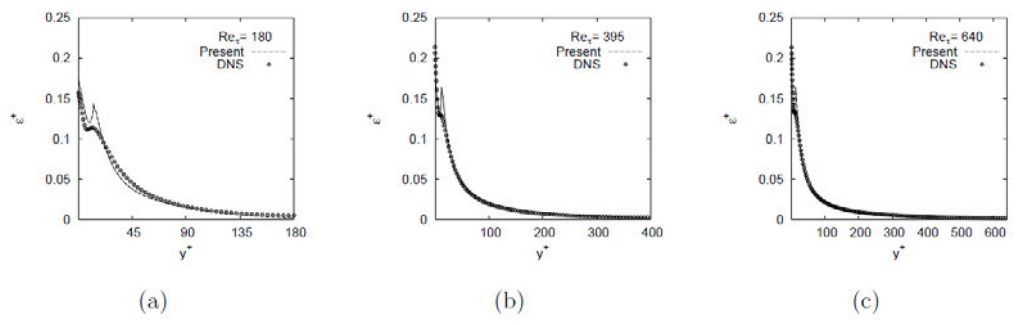


Figure 4: Dissipation-rate profiles for fully-developed turbulent channel flow.

Predicted profiles of the velocity and turbulent shear-stress from independent turbulence models are illustrated in Figures 1 and 2, respectively. It seems likely that indistinguishable predictive performances pertaining to both models are obtained. As can be seen, two turbulence models make pretty good correspondence with *DNS* data in both regions, comprising the linear boundary layer and wake defect layer. Present model performances are further assessed with turbulent kinetic energy  $k^+$  and dissipation-rate  $\varepsilon^+$  profiles as shown in Figures 3 and 4, respectively. Note-worthily, reasonable agreement of the current model with *DNS* data is visible without having transport and diffusion effects of the turbulent kinetic energy and dissipation-rate. It appears that the near-wall  $k^+$ -profile is qualitatively well reproduced and the maximum magnitude of  $\varepsilon^+$  is captured in the wall-vicinity, as dictated by *DNS* and experimental data.

## 4 Conclusions

A compatible eddy-viscosity coefficient is introduced with the current model, the potential importance of which is not obvious since only a fully-developed turbulent channel flow case (e.g., simple shear flow case) is computed for validation. However, it is believed that the modification is profoundly convenient to account for strong flow in-homogeneity and near-wall turbulence and therefore, it can enhance the model competency in speculating complex separated and reattaching flows to a greater extent. Articulately, the link to the  $k - \varepsilon$  model with the present model is retrieved and likely to be extendable toward a non-linear algebraic Reynolds stress model. Intuitively, the present formulation may accommodate a variable eddy-viscosity coefficient for an *LES* or a *DES* method.

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